

Linear ball bushings



Linear ball bushings

Load rating

Basic dynamic load rating

This term is arrived at based on an evaluation of a number of identical linear systems individually run in the same conditions, if 90% of them can run with the load (with a constant value in a constant direction) for a distance of 50 km without damage caused by rolling fatique. This is the basis of the rating.

Allowable static moment

This term defines the allowable limit value of static moment load, with reference to the amount of permanent deformation similar to that used for evalution of basic rated load (Co).

Static safety factor

This factor is used based on the application condition as shown in Table 1.

Basic static load rating

This term defines a static load such that, at the contacting position where the maximum stress is exercised, the sum of the permanent deformation of the rolling elements and that of the rolling plane is 0,0001 time of the diameter of the rolling elements.

Static safety factors

Table 1.

Condition of use	Low limit of fs
When the shaft has less deflection and shock	1 to 2
When elastic deformation should be considered	
whith respect to pinch load	2 to 4
When the equipment is subject to vibration and impacts	3 to 5

Rating Life

Rating life of the linear system

As long as linear system reciprocates while being loaded, continous stress acts on the linear system to couse flaking on the rolling bodies and planes because of material fatigue. The travelling distance of linear system until the fist flaking occurs is called the life of the system. The life of the dimensions, structure, material, heat treatment and processing method, when used in the same conditions. This variation is brought about from the essential variations in the material fatigue itself. The rating life defined bellow is used as an index for the life expectancy of the linear system.

Rating life

Rating life is the total travelling distance that 90% of a group of systems of the same size can reach without causing any flaking when they operate under the same conditions.

The rating life can be obtained from the following equation with the basic dynamic load rating and the load on the linear system:

For ball type:

$$L = \left(\frac{C}{P}\right)^3 50,$$

where:

L - rating life, km,

C - basic dynamic load rating N

P -load, N.

Consideration and influence of vibration impact loads and distribution of load should be taken into account when designing a linear motion system. It is difficult to calculate the actual load. The rating life is also affected by the operating temperature. In these conditions, the expression (1) is arranged as follows:

For ball type:

$$L = \left(\frac{f_H \stackrel{3}{\times} f_T \times f_C}{f_W \times P}\right) \times 50,$$



where:

-rating life, km, ı

-hardness factor (see figure 1), fн

Ċ -basic dynamic load rating, N.

fт -temperatuer coefficient (see figure 2). P

-load, N.

fc -contact coefficient (see table 2).

-load coefficient (see table 3). fw

The rating life in hours can be calculated by obtaining the travelling distance per unit time. The rating life in hours can be obtained from the following expression when the stroke length and the number of strokes are constant:

$$L_h = \frac{L \times 10^3}{2 \, I_s \times n_1 \times 60} \,,$$
 where:

Lh -rating life in hours, hr,

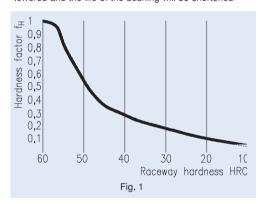
-stroke length, m, ls

L -rating life, km,

-no of strokes per minute, cpm.

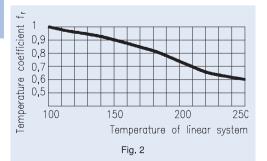
Hardness factor

The shaft be sufficiently hardened when a linear bushing is used. If not properly hardened, permissible load is lowered and the life of the bushing will be shortened



Temperature coefficient

If the temperature of the linear system exceeds 100°C, Hardness of the linear system and the shaft lowers to decresure the permissible load compared to that of the linear system used at room temperature rise shortens the rating life.



Contact coefficient

Generally two or more linear bushings are used on one shaft. Thus, the load on each linear system differs depending on each precessing accuracy. Because the linear bushings are not loaded equally, the number of linear bushings per shaft changes the permissible load of system.

Contact coefficient

Table 2

Number of linear systems per shaft	Contact coefficient fc
1	1,00
2	0,81
3	0,72
4	0,66
5	0,61

Load coefficient

When calculating the load on the linear system, it is necessary to accurately obtain object weight, inertial force based on motion speed, moment load, and each transition as time passes. However, it is difficult to calculate those values accurately because reciprocating motion involves the repetition of start and stop as well as vibration and impact. A more practical approach is to obtain the load coefficient by taking the actual operating conditions into account.

Load coefficient

Table 3

Operating Conditions	fw
Operation at low speed(15m/min. or less) without impulsive shock from outside	1,0 to 1,5
Operation at intermediate speed (60 m/min. or less) without impulsive shock	1,5 to 2,0
Operation at high speed (over 60 m/min.) With impulsive shock from outside	2,0 to 3,5

Frictional resistance

The static frictional resistance of the MTK linear system is so low as to be only slightly different from the kinetic frictional resistance, enabling smooth linear movement from low to high speeds. In general, the friction resistance is expressed by the following equation.

$$F = \mu W + f$$

where:

F -frictional resistance.

-coefficient of friction.

w -load weight,

-sealing resistance.

The frictional resistance of each MTK linear system depends on the model, load weight, speed, and lubricant. The sealing resistance depends on lip interference and lubricant, regardless of the load weigh. The sealing resistance of one linear system is about 200 to 500 gf. The coefficient of friction depends on the load weight, moment load, and preload. Table 6 shows the coefficient of kinetic friction of each type of linear system which has been instaled and lubricated properly and applied with normal load (P/C= 0.2)

Coefficient of linear system friction

Table 4

Linear System Type	Models	Coefficient of Friction
Linear Bushing	LM LME LMB	0,002 to 0,003

Ambient working temperature

The ambient working temperature range for each MTK linear system depends on the model. Consult MTK on use outside the recommended temperature range.

Temperature conversion equation:

$$C = \frac{5}{9} (F-32)$$

$$F = 32 + \frac{9}{5}C$$

Ambient working temperature

Table 5

Linear System Type	Models	Ambient Working Temperature
Linear Bushing	LM LME LMB	-20 to 80°C

Lubrication and dust prevention

Using MTK linear systems without lubrication increases the abrasion of the rolling elements, shortening the life span. The MTK linear systems, therefore require appropriate lubrication. For lubrication MTK recommends turbine oil conforming to ISO Standards G32 to G68 or lithium base soap grease no. 2. Some MTK linear systems are sealed to block dust out and seal lubricant in. If used in a harsh or corrosive environment, however, apply a protective cover to the part involving linear motion.

Structure and features

The MTK linear bushing consists of an outer cylinder, ball retainer, balls and two end rings. The ball retainer which holds the balls in the recirculating trucks inheld inside the outer cylinder by end rings.

Those parts are assembled to optimize their required func-

tions

The outer cylinder is maintained sufficient hardness by heat treatment, therefore if ensures the bushings projected travel life and satisfactory durability.

The ball retainer is made from synthetics to reduce running noise.

High precision and rigidity

The MTK linear bushing is reduced from a solid steel outer cylinder and incorporates an industrial strength resin retainer

Ease of assembly

The standard type of MTK linear bushing can be loaded from any direction. Precision control is possible using only the shaft supporter, and the mounting surface can be machined easily.

Ease of replacement

MTK linear bushing of each type are completely interchangeable because of their standardized dimensions and strict precision control. Replacement because of wear or damage is therefore easy and accurate.

Variety of types

MTK offers a full line of linear bushing: The standard, integral single - retainer closed type, the clearance adjustable type and the open types. The user can choose from among these according to the application requirements to be met

Linear ball bushing designation

Designation

Group I Group II Group III Group IV Nominal Modification Seal Type shaft diameter

Example: LM 25 UU AJ

Type:

LM -metric dimension series most widely used

in Japan.

LME -metric dimension series generally used in

LMB -inch dimension series used mainly in USA.

Modification:

No entry -standard type, AJ -adjustable type, OP -open type.

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Seal:

-no seal.

No entry

П -seal on one side. UU -seals on both sides

Tolerance

Note that precision of inscribed circle diameters and outside diameters for the clearance adjustable type (...-AJ) and the open type (...-OP) indicates the value obtained before the corresponding type is subjected to cutting process.

Load rating and life expectancy

The life of a linear bushing can be obtained from the following equation with the basic dynamic load rating and the load applied to the bush:

$$L = \left(\frac{f_{H} \times f_{T} \times f_{C}}{f_{W} \times P} \times \frac{C}{P}\right)^{3} \times 50, \tag{1}$$

where:

L -ratedlife, km,

С -basic dynamic load rating, N,

-working load, N, fw -load coefficient.

-hardness factor (see page 340), fн

-temperature coefficient (see page 340),

-contact coefficient (see page 340).

The lifespan of a linear bushing in hours can be obtained by calculating the travelling distance per unit time.

The lifespan can be obtained from the following equation. if the stroke length and the number of strokes are constant:

$$L_h = \left(\frac{L \times 10^3}{2 \times I_s \times n_1 \times 60}\right), \tag{2}$$

where:

Lh -lifespan, hr,

-stroke length, m, ls

-rated life, km, L

-number of strokes per minute, cpm.

Relation between ball circuits and load rating

The MTK linear bushing includes ball circuits that are spaced equally and circumferentially. The load rating varies according to the loaded position on the circumference.

The value in the dimension table indicates the load rating when the load is placed on top of one ball circuit. If the MTK linear bushing is used with two ball circuits loaded uniformly, the load rating will be greater. The following table shows the values by the number of ball circuits in such cases:

Tabel 6

Row			Number of rows		
position load ratio	3	4	5	6	8
Row position load ratio	P ₀		P _D P _D P _D		P ₁
	Q ₁ = P ₀	$Q_1 = P_0$	$Q1 = 1,106P_0$	$Q1 = 1,354P_0$	$Q1 = 1,841P_0$
Row position	P ₀	P ₀	P ₀	P ₀	P ₀ P ₀
	$Q_0 = P_0$	$Q0 = 1,414P_0$	$Q0 = 1,618P_0$	Q0 =1,732 P ₀	Q0 = 2,052P ₀
Load ratio	$Q_0/Q_1 = 1$	$Q_0/Q_1 = 1,414$	$Q_0/Q_1 = 1,463$	$Q_0/Q_1 = 1,280$	$Q_0/Q_1 = 1,115$

Sample calculations

Obtaining the rated life and lifespan the MTK linear bushing used in the following conditions:

Linear bushing LM20
Stroke length 50 mm
Number of strokes per minute 50 cpm
Load per bush 490 N

The basic dynamic load rating of the linear bushing is 882N from the dimension table. From equation (1) therefore, the rated life is obtained as follows:

$$L = \left(\frac{f_{H} \times f_{T} \times f_{C}}{f_{W} \times P} \times \frac{C}{P}\right)^{3} \times \ 50 = \left(\frac{882}{490}\right)^{3} \times 50 = 292 \, \text{km},$$

where:

$$F_H = f_T = f_C = f_W = 1.0$$

From equation (2), the lifespan is obtained as followes:

$$L_h = \left(\frac{L \times 10^3}{2 \times e_s \times n_1 \times 60}\right) = \left(\frac{292 \times 10^3}{2 \times 0.05 \times 50 \times 60}\right) = 973 \text{ hr}$$

Selecting the linear bushing type satisfying the following conditions:

Number of linear bushing used 4
Stroke length 1 m
Traveling speed 10 m/min
Number of strokes per minute Lifespan 10 hr
Total load 980 N

From equation (2), the travelling distance within the lifespan is obtained as follows

$$L = 2 \times I_S \times n_1 \times 60 \times L_h = 6000 \text{ km}$$

From ecuation (1), the basic dynamic load rating is obtained as follows:

$$C = \sqrt[3]{\frac{L}{50}} \times \left(\frac{f_W}{f_H \times f_T \times f_C}\right) \times P = 1492 \text{ N}$$

Assume the following with a pair of shafts each with two linear bushings:

$$f_C = 0.81$$
, $f_W = f_T = f_H = 1$

As a result, LM30 is selected from the dimension table as the MTK linear bushing type satisfying the value of C.

Clearance and fit

When a standard-type MTK linear bushing is used with a shaft, inadequate clearance, adjustment may cause early bush failure and/or poor, rough traveling. The clearence adjustable linear bush and open linear bush can be clearence adjusted when assembled in the housing which can control the outside cylinder diameter. However, too much clearance adjustment increases the deformation of the outside cylinder, to affect its precision and life. Therefore, the appropriate clearance between the bush and shaft, and clearance

between the bush and housing are required according to the application. Table 7 shows recommended fit of the bush:

Table 7

Division Model	Shaft Normal fit High class	Transi- tional	Housing Loose fit	Tight fit
LM LMB	g6	h6	H7	J7

Note. The clearance may be zero or negative. Please attention the movement.

Shaft and housing

To optimize performance of the MTK linear bushing high precision of the shaft and housing is required.

Shaft

The rolling balls in the MTK linear bushing are in point contact with the shaft surface. Therefore, the shaft dimensions, tolerance, surface finish and hardness greatly affect the travelling performance of the bush. The shaft should be manufactered with due attention to the following points:

- Since the surface finish critically affects smooth rolling of balls, grind the shaft at 1,5 S or better.
- The best hardness of the shaft is HRC 60 to 64; Hardness less than HRC 60 decreases the life considerably, and hence reduces the permissible load. On the other hand, hardness over HRC 64 accelerates ball wear.
- The shaft diameter for the clearance adjustable linear bush and open linear bush should as much as possible be of the lower value of the inscribed circle diameter in the specification table. Do not set the shaft diameter to the upper value.
- Zero clearance or negative clearance increases the frictional resistance slightly. If the negative clearance is too tight, the deformation of the outside cylinder will become larger, to shorten the bush life.

Housing

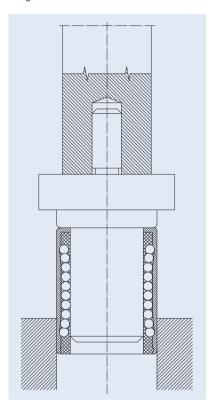
There is a wide range of housings differing in design, machining and mounting. For the fitness and shapes of housings see in table 8 and the following section on mounting.

Mounting

When inserting the linear bush into the bushing do not hit the linear bush on the side ring holding the retainer but apply the cylinder circumference with a proper jig and push the linear bush into the housing by hand or lightly knock it in. In inserting the shaft after mounting the bush,

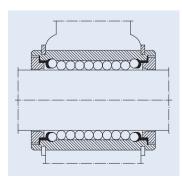


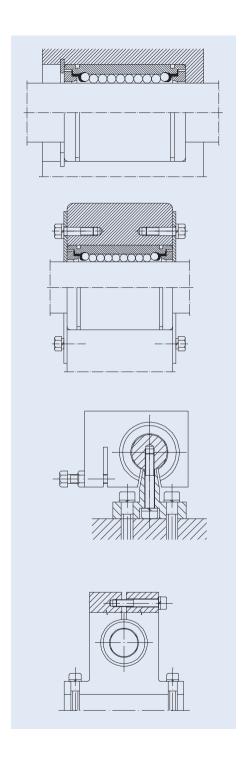
be careful not to shock the balls. Note that if two shafts are used in parallel, the parallelism is the most important factor to assure the smooth linear movement. Take care in setting the shafts.



Exemples of mounting

The popular way to mount a linear bush is to operate it with an appropriate interference. It is recommended, however, to make a loose fit in principle because otherwise precision is apt to be minimized. The following examples show assembling of the inserted bush in terms of designing and mounting for reference.





MTK ball bushing interchangeability list

Ball bushing compact type

MTK	NTN	STAR	INA	SKF	FAG
KH	KH	0658 - 000	KH	LBBR LBBS	LNA LFA
KHPP	KHLL	0658 - 240	KHPP	LBBR2LS	LNA2RS

Ball bushing resin retainer

MTK	NB	INA	SKF	THK	IKO	THOMSON	EASE
LME	KBG	KB	LBAR/LBCR	LME	LBE	MA M	SDE
LMEUU	KBGUU	KBPP	LBAR/LBCR2LS	LMEUU	LBEUU	MA MWW	SDEUU
LMEAJ	KBGAJ	KBS	LBAS	LMEAJ	LBEAJ	MA MADJ	SDEAJ
LMEUUAJ	KBGUUAJ	KBSPP	LBAS2LS	LMEUUAJ	LBEUUAJ	MA MADJ WW	SDEUUAJ
LMEOP	KBGOP	KBO	LBAT/LBCT	LMEOP	LBEOP	MA MOPN	SDEOP
LMEUUOP	KBGUUOP	KBOPP	LBAT/LBCT2LS	LMEUUOP	LBEUUOP	MA MOPN WW	SDEUUOP

The above types are metric dimension series generally used in Europe.

LM SMG LM SDM LMB SWG LMUU SMGUU LMUU SDMUU LMBUU SWGUU LMAJ SMGAJ LMAJ SDMAJ LMBAJ SWGAJ LMUUAJ SMGUUAJ LMUUAJN SDMUUAJ LMBAJ SWGAJ LMOP SMGOP LMOP SDMUUAJ LMBOP SWGUUAJ LMOP SMGUUOP LMUUOP SDMUUOP LMBOP SWGUUOP	LMBOP SDB.	AJ UUAJ

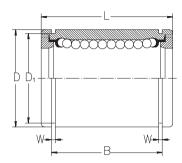
The above types are metric dimension series generally used in Japan and other countries.

The above types are inch dimension series generally used in US.

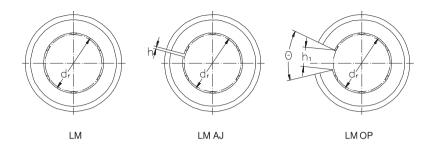
Standard type	page 346 page 348 page 350	
Open type	page 346 page 348 page 350	One ball circuit (50° - 80°) is removed to allow an opening slot to fit over rail supports.
Adjustable type	page 346 page 348 page 350	This type has a slot in the outside cylinder. This design allows for clearance adjustment.
Drawn cup type	page 345	This type linear ball bushings consist of thin walled drawn cups, plastic cages and grade 10 steel balls. Bushings are available with seals at one or both ends.

MTK+

Linear ball bushing



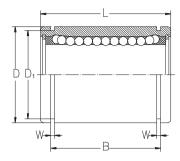
Nominal part I Standard type	n o. Seal type	Ball circuit	Weight	Adjustable type	Open type	Nominal s Tolerance	shaft diameter
_	_		gr	_	_	mm	
LM 5	LM 5UU	4	4	_	_	5 ⁰	-0,008
LM 6	LM 6UU	4	8	LM 6 AJ	_	6 0	-0,009
LM 8S	LM 8SUU	4	11	LM 8S AJ	_	8	
LM 8	LM 8UU	4	16	LM 8 AJ	_	8	
LM 10	LM 10UU	4	30	LM 10 AJ	_	10	
LM 12	LM 12UU	4	31,5	LM 12 AJ	LM 12 OP	12	
LM 13	LM 13UU	4	43	LM 13 AJ	LM 13 OP	13	
LM 16	LM 16UU	4	69	LM 16 AJ	LM 16 OP	16	
LM 20	LM 20UU	5	87	LM 20 AJ	LM 20 OP	20	-0,010
LM 25	LM 25UU	6	220	LM 25 AJ	LM 25 OP	25	
LM 30	LM 30UU	6	250	LM 30 AJ	LM 30 OP	30	
LM 35	LM 35UU	6	390	LM 35 AJ	LM 35 OP	35	-0,012
LM 40	LM 40UU	6	585	LM 40 AJ	LM 40 OP	40	
LM 50	LM 50UU	6	1580	LM 50 AJ	LM 50 OP	50	
LM 60	LM 60UU	6	2000	LM 60 AJ	LM 60 OP	60 ⁰	-0,015



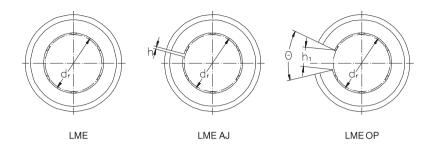
		nsions and LTolerance		W	D1	Н	h1		Eccen- tricity max	Radial clearance max	Basic Load C	Rating C ₀	Nominal part no.
mm									μm		kN		
10-0,0		15-0,012	10,2-0,2	1,1	9,6	_	_	_	8	-3	0,17	0,21	LM 5
12-0,0		19 _{-0,02} ⁰	13,5-0,2	1,1	11,5	1	_	_	12	-5	0,21	0,27	LM 6
15-0,0		17-0,02	11,5-0,2	1,1	14,3	1	_	_	12	- 5	0,18	0,23	LM 8S
15-0,0		24-0,02	17,5-0,2	1,1	14,3	1	_	_	12	- 5	0,27	0,41	LM 8
19-0,0	0 13	29-0,02	22-0,2	1,3	18	1			12	-5	0,38	0,56	LM 10
21-0,0		30-0,02	23-0,2	1,3	20	1,5	8	80°	12	-5	0,42	0,61	LM 12
23-0,0		32-0,02	23-0,2	1,3	22	1,5	9	80°	12	-7	0,52	0,79	LM 13
28-0,0		37-0,02	26,5-0,2	1,6	27	1,5	11	80°	12	-7	0,79	1,2	LM 16
32-0,0	16	42-0,02	30,5-0,2	1,6	30,5	1,5	11	60°	15	-9	0,88	1,4	LM 20
40-0,0	16	59 _{-0,03}	41-0,3	1,85	38	2	12	50°	15	-9	1	1,6	LM 25
45-0,0		64-0,03	44,5-0,3	1,85	43	2,5	15	50°	15	-9	1,6	2,8	LM 30
52-0,0		70-0,03	49,5-0,3	2,1	49	2,5	17	50°	20	-13	1,7	3,2	LM 35
60-0,0		80-0,03	60,5-0,3	2,1	57	3	20	50°	20	-13	2,2	4,1	LM 40
70-0,0	0 22 0	100-0,03	74-0,3	2,6	76,5	3	25	50°	20	-13	3,9	8,1	LM 50
80-0,0		110 _{-0,03}	85-0,3	3,15	86,5	3	30	50°	25	-16	4,8	10,2	LM 60

LM<Built- in synthetics resin retainer> This type is a metric dimension series widely used in Japan and other countries

Linear ball bushing



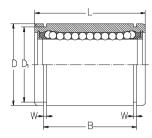
Nominal pa Standard type	art no. Seal type	Ball circuit	Weight	Adjustable type	Open type	Nominal shaft diameter Tolerance
_	_		gr	_	_	mm
LME 5	LME 5UU	3	11	LME 5 AJ	_	5 0 +0,008
LME 8	LME 8UU	4	20	LME 8 AJ	_	8
LME 12	LME 12UU	4	41	LME 12 AJ	LME 12 OP	12
LME 16	LME 16UU	4	57	LME 16 AJ	LME 16 OP	16 -0,001
LME 20	LME 20UU	5	91	LME 20 AJ	LME 20 OP	20
LME 25	LME 25UU	6	215	LME 25 AJ	LME 25 OP	+0,011 25 -0,001
LME 30	LME 30UU	6	325	LME 30 AJ	LM E 30 OP	30
LME 40	LME 40UU	6	705	LME 40 AJ	LME 40 OP	+0,013 40 -0,002
LME 50	LME 50UU	6	1130	LME 50 AJ	LM E 50 OP	50
LME 60	LME 60UU	6	2220	LME 60 AJ	LM E 60 OP	60



Major dime DTolerance		d tolerance BTolerance	w	D1	Н	h1		Eccen- tricity max	Radial clearance max	Basic e load C	Rating C ₀	Nominal part no.
mm								μm		kgF		
12-0,008	22-0,02	14,5-0,2	1,1	11,5	1	_	_	12	-5	21	27	LME 5
16 _{-0,008}	25-0,02	16,5-0,2	1,1	15,2	1	_	_	12	- 5	21	41	LME 8
22-0,009	32-0,02	22,9-0,2	1,3	21	1,5	7,5	78°	12	-7	52	79	LME 12
26-0,009	36-0,02	24,9-0,2	1,3	24,9	1,5	10	78°	12	-7	59	91	LME 16
32-0,011	45-0,02	31,5-0,2	1,6	30,3	2	10	60°	15	-9	88	140	LME 20
40-0,011	580-0,03	44,1-0,3	1,85	37,5	2	12,5	60°	15	-9	100	160	LME 25
47-0,011	68-0,03	52,1-0,3	1,85	44,5	2	12,5	50°	15	-9	160	280	LME 30
62-0,013	80-0,03	60,6-0,3	2,15	59	3	16,8	50°	17	-13	220	410	LME 40
75 _{-0,013}	100-0,03	77,6-0,3	2,65	72	3	21	50°	17	-13	390	810	LME 50
90-0,015	125-0,04	101,7-0,4	3,15	86,5	3	27,2	54°	20	-16	480	1020	LME 60

LME<Built- in synthetics resin retainer> This type is a metric dimension series generally used in Europe.

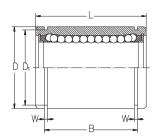
Linear ball bushing



Nominal diameter	Nominal p	art number					Nominal diameter		Major dimensions and	
ulameter	Standard	Standard Seal Ball Weight Adjustable type circuit type		Open type	diameter		tolerance			
	туре	туре	Circuit	,	туре	туре		Tolerance	D Tole	rance
inch/mm				kg			inch/mm	ı		
1/4 6,350	LMB 4	LMB 4UU	4	0,008	LMB 4 AJ	_	0,250 6,350	0 -0,0040	0,5000 12,700	0 -0,00045
										0 -0,011
3/8 9,525	LMB 6	LMB 6UU	4	0,014	LMB 6 AJ	_	0,3750 9,525		0,6250 15,875	0 -0,00050
1/2 12,700	LMB 8	LMB 8UU	4	0,037	LMB 8 AJ	LMB 8 OP	0,5000 12,700	0 -0,0090	0,8750 22,225	
5/8 15,875	LMB 10	LMB 10U	J 4	0,076	LMB 10 AJ	LMB 10 OP	0,625 15,875		1,1250 28,575	
3/4 19,050	LMB 12	LMB 12U	J 5	0,095	LMB 12 AJ	LMB 12 OP	0,7500 19,050	0 -0,0040	1,2500 31,750	0 -0,00065
1 25,400	LMB 16	LMB 16U	J 6	0,200	LMB 16 AJ	LMB 16 OP	1,0000 25,400		1,5625 39,688	
1-1/4 31,750	LMB 20	LMB 20U	J 6	0,440	LMB 20 AJ	LMB 20 OP	1,2500 31,750	0 -0,0050	2,0000 50,800	
1-1/2 38,000	LMB 24	LMB 24UU	J 6	0,670	LMB 24 AJ	LMB 24 OP	1,5000 38,100		2,3750 60,325	
2 50,800	LMB 32	LMB 32U	J 6	0,114	LMB 32 AJ	LMB 32 OP	2,0000 50,800	0 -0,010	3,0000 76,200	0 -0,00090 0 -0.022



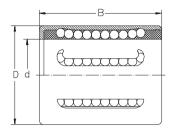
LM<Built- in synthetics Resin Retainer> This type is a metric dimension series widely used in Japan and other countries

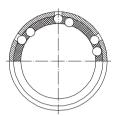


	Tolerance	В	Tolerance	W	D ₁	h	h ₁		Eccen- tricity max	Radial clearance max	Basic e load rating C C ₀	Nominal part no.
0,7500		0,5110	0	0.390	0.4697	0.04			0.0005	-0.0001	206 265	LMB 4
	-0,008	12,98	-0,008	0,992	0,4687 11,906	1		_	0,0005 12	-3	200 200	LIVID 4
	0 -0,200		0 -0,200									
0,8750 22,225		0,6358 16,15		0,390 0,992	0,5880 14,935	0,04 1	_	_	0,0005 12	-0,0001 -3	225 314	LMB 6
1,2500 31,750		0,9625 24,46		0,0459 1,168	0,8209 20,853	0,06 1,5	0,34 7,9375	80°	0,0005 12	-0,0001 -4	510 764	LMB 8
1,5000 38,100		1,1039 28,04		0,0559 1,422	1,0590 26,899	0,06 1,5	0,375 9,525	80°	0,0005 12	-0,0001 -4	774 1180	LMB 10
1,6250 41,275		1,1657 29,61		0,0559 1,422	1,1760 29,870	0,06 1,5	0,4375 11,1125	60°	0,0006 15	-0,0002 -6	862 1370	LMB 12
2,2500 57,150	0 -0,012	1,7547 44,57	0 -0,012	0,0679 1,727	1,4687 37,306	0,06 1,5	0,5625 14,2875	50°	0,0006 15	-0,0002 -6	980 1570	LMB 16
2,6250 66,675	0 -0,300	2,0047 50,92	0 -0,300	0,0679 1,727	1,8859 47,904	0,10 2,5	0,625 15,875	50°	0,0008 20	-0,0003 -8	1570 2740	LMB 20
3,0000 76,200		2,4118 61,26		0,0859 2,184	2,2389 56,870	0,12 3	0,75 19,05	50°	0,0008 20	-0,0003 -8	2180 4020	LMB 24
4,0000 101,600	0	3,1917 81,07		0,1029 2,616	2,8379 72,085	0,12 3	1,0 25,40	50°	0,0010 25	-0,0005 -13	3820 7940	LMB 32
	0 -0,022											



Standard linear ball bushing Steel drawn cup/cage plastic





Dime	nsions D	В	Load capac dyn.	city stat.	Designation	Weight
mm			N		_	g
6 8 10 12 14	12 15 17 19 21	22 24 26 28 28	400 435 500 620 620	239 280 370 510 520	KH 0622 KH 0824 KH 1026 KH 1228 KH 1428	7 12 14,5 18,5 20,5
16 20 25 30 40	24 28 35 40 52	30 30 40 50 60	800 950 1990 2800 4400	620 790 1670 2700 4450	KH 1630 KH 2030 KH 2540 KH 3050 KH 4060	27,5 32,5 66 95 182
50	62	70	5500	6300	KH 5070	252